

## Damped / Driven Linear Oscillators from 325

EOM :  $\ddot{x} + 2\beta\dot{x} + \omega_0^2 x = f(t)$  where  $\vec{F}_{\text{restore}} = -k\vec{x} \equiv -m\omega_0^2 \vec{x}$ ,  $\vec{F}_{\text{damp}} = -m(2\beta)\vec{v}$ ,  $\vec{F}_{\text{drive}} = m f(t)$

Damping Cases : Critical damping :  $\beta = \omega_0 \equiv \sqrt{\frac{k}{m}}$ , Underdamping:  $\beta < \omega_0$ , Overdamping:  $\beta > \omega_0$

## Coupled Linear Oscillators

$$\mathbf{M}\ddot{\vec{q}} = -\mathbf{K}\vec{q} \quad T = \frac{1}{2}\mathbf{M}_{ij}\dot{q}_i\dot{q}_j \quad U = \frac{1}{2}\mathbf{K}_{ij}q_iq_j \quad \mathbf{M}_{ij} = \left. \frac{\partial^2 T}{\partial \dot{q}_i \partial \dot{q}_j} \right|_{\dot{q}=0} \quad \mathbf{K}_{ij} = \left. \frac{\partial^2 U}{\partial q_i \partial q_j} \right|_{q=0}$$

Transverse oscillations of taut string:  $k_T = \left. \frac{\text{tension}}{\text{length}} \right|_{\text{equilib}}$

## Inner Product Space description and Normal Coordinates

\*  $\vec{x}$  = column vector of generalized coord

- **Space** :  $|\vec{x}(t)\rangle \equiv$  solutions of a linear oscillator system
- **Inner Product** :  $\langle \vec{y} | \vec{x} \rangle \equiv \vec{y}^T \mathbf{M} \vec{x}$  and associated **magnitude** :  $|\vec{x}|^2 \equiv \langle \vec{x} | \vec{x} \rangle$
- **Basis** :  $|\hat{a}_m\rangle$  of eigenvectors defined by  $\mathbf{K}\vec{a}_m = \omega_m^2 \mathbf{M}\vec{a}_m$  and normalization  $\hat{a}_m \equiv \vec{a}_m / |\vec{a}_m|$
- **Basis is Orthonormal** :  $\langle \hat{a}_n | \hat{a}_m \rangle = \delta_{nm}$
- **Completeness** for  $\vec{x}(t)$  and **Normal Coordinates**  $\xi_m$  :

$$\xi_m \text{ is the component of } \vec{x} \text{ along mode } m : |\vec{x}(t)\rangle = \sum_{\text{modes } m} |\hat{a}_m\rangle \langle \hat{a}_m | \vec{x}(t)\rangle \equiv \sum_{\text{modes } m} \hat{a}_m \xi_m(t)$$

$$\xi_m \text{ is projected out of } \vec{x} \text{ by : } \xi_m(t) = \langle \hat{a}_m | \vec{x}(t)\rangle = A_m \cos(\omega_m t - \delta_m)$$

- **Transformation** between  $x$ -space and  $\xi$ -space :

$$\text{vectors : } \vec{\xi} = \mathbf{R} \vec{x} \quad \vec{x} = \mathbf{R}^{-1} \vec{\xi} \quad \mathbf{R}^{-1} = \begin{pmatrix} | & | & \\ \hat{a}_1 & \hat{a}_2 & \dots \\ | & | & \end{pmatrix} \quad \mathbf{R} = (\mathbf{R}^{-1})^T \mathbf{M}$$

$$\text{matrices : } \mathbf{M}^\xi = (\mathbf{R}^{-1})^T \mathbf{M} \mathbf{R}^{-1} \rightarrow \mathbf{M}_{mn}^\xi = \delta_{mn} \quad \& \quad \mathbf{K}_{mn}^\xi = \omega_m^2 \delta_{mn}$$

## Lagrangian Mech from 325 \* Gen. coord $q_i$ must be indep

Principle of Least Action :

$$S \equiv \int_{t_1, \vec{q}_1}^{t_1, \vec{q}_1} dt L(q_i, \dot{q}_i, t) \quad \delta S = 0 \rightarrow \frac{\partial L}{\partial q_i} = \frac{d}{dt} \left( \frac{\partial L}{\partial \dot{q}_i} \right) \text{ for each } q_i$$

$$L = T - U \rightarrow \delta S = 0 @ \text{ true } \{q_i(t)\}$$

$$\text{Gen. force } Q_i \equiv \frac{\partial L}{\partial q_i}, \text{ momentum } p_i \equiv \frac{\partial L}{\partial \dot{q}_i}$$

$$H \equiv \dot{q}_i (\partial L / \partial \dot{q}_i) - L \text{ conserved when } \partial L / \partial t = 0$$

$$H \equiv p_i \dot{q}_i - L \text{ equals } T+U \text{ when } \vec{r}_a = \vec{r}_a(q_i)$$